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## Power handling of the JET ITER-like wall

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# Power handling of the JET ITER-like wall

G Arnoux<sup>1</sup>, I Balboa<sup>1</sup>, M Clever<sup>2</sup>, S Devaux<sup>3</sup>, P De Vries<sup>4</sup>, T Eich<sup>3</sup>,  
M Firdaouss<sup>5</sup>, S Jachmich<sup>6</sup>, M Lehnen<sup>2</sup>, P J Lomas<sup>1</sup>, G F Matthews<sup>1</sup>,  
Ph Mertens<sup>3</sup>, I Nunes<sup>7</sup>, V Riccardo<sup>1</sup>, C Ruset<sup>8</sup>, B Sieglin<sup>3</sup>, D F Valcárcel<sup>7</sup>,  
J Wilson<sup>1</sup>, K-D Zastrow<sup>1</sup> and JET-EFDA contributors<sup>9,10</sup>

<sup>1</sup> EURATOM/CCFE Fusion Association, Culham Science Centre, Abingdon, Oxon OX14 3DB, UK

<sup>2</sup> Forschungszentrum Jülich, Institute of Energy Research—Plasma Physics, EURATOM Association, D-52425 Jülich, Germany

<sup>3</sup> Max-Planck-Institut für Plasmaphysik, EURATOM-Assoziaton, D-85748 Garching, Germany

<sup>4</sup> FOM Institute DIFFER PO Box 1207, NL-3430 BE Nieuwegein, The Netherlands

<sup>5</sup> Association EURATOM-CEA, CEA/DSM/IRFM, Cadarache, F-13108 Saint Paul Lez Durance, France

<sup>6</sup> Association 'EURATOM—Belgian State' Laboratory for Plasma Physics, Koninklijke Militaire School—Ecole Royale Militaire Renaissancelaan, 30 Avenue de la Renaissance, B-1000 Brussels, Belgium

<sup>7</sup> Associao EURATOM/IST, Instituto de Plasmas e Fuso Nuclear, Instituto Superior Técnico, Av Rovisco Pais, 1049-001 Lisbon, Portugal

<sup>8</sup> National Institute for Laser, Plasma and Radiation Physics, Association Euratom-MEDC, Bucharest, Romania

<sup>9</sup> JET-EFDA, Culham Science Centre, Abingdon OX14 3DB, UK

E-mail: [gilles.arnoux@ccfe.ac.uk](mailto:gilles.arnoux@ccfe.ac.uk)

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## Abstract

The ITER-like wall (ILW) at JET provides a unique opportunity to study the combination of material (beryllium and tungsten) that will be used for the plasma facing components (PFCs) in ITER. Both the limiters (Be) and divertor (CFC W coated and bulk W) have been designed to maximize their power handling capability. During the last experimental campaign (October 2010–July 2011) this capability has been assessed and even challenged in the case of the Be wall. The Be limiters' power handling capability ( $19 \text{ MW m}^{-2} \text{ s}^{-1/2}$ ), predicted with a simple model, has been proven to be robust by the experiments despite an unexpected power load pattern. This capability has been pushed to its limit leading to Be melt events, which revealed that the power load is toroidally asymmetric. The protection system of the ILW did not prevent melt events mainly because the protection strategy relies on the assumption that the power load is toroidally symmetric. The bulk W divertor target performed as predicted. Operations were constrained by: (i) an energy load limit ( $60 \text{ MJ m}^{-2}$ ); (ii) the limited number of cycles of the surface temperature above  $1200^\circ\text{C}$  in order to prevent thermal fatigue. This latter limit has been exceeded about 300 times and no signs of damage or thermal fatigue have been observed by the photogrammetric survey.

Keywords: divertor, power exhaust, tokamak, plasma facing components, plasma wall interaction

(Some figures may appear in colour only in the online journal)

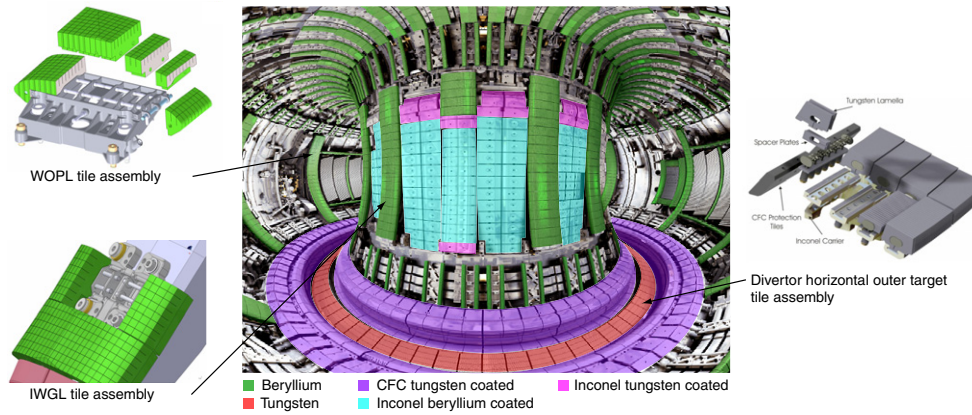
## 1. Introduction

The ITER-like wall (ILW) at JET is of the same combination of plasma facing materials as is foreseen for the active operation in ITER: beryllium for the first wall and tungsten

in the divertor. The ILW provides a unique opportunity to test the power handling capability of the different plasma facing components (PFCs) under relevant heat loads conditions due to plasma wall interaction.

Be has a melting temperature of  $1257^\circ\text{C}$ , which typically is reached in 10 s with a heat load of  $6 \text{ MW m}^{-2}$  ( $19 \text{ MW m}^{-2} \text{ s}^{-1/2}$ ). The W has a much higher melting

<sup>10</sup> See the appendix of Romanelli F *et al* 2012 *Proc. 24th IAEA Fusion Energy Conf.* (San Diego, CA, 2012) OV/1–3.



**Figure 1.** Overview of the JET ILW PFCs. The first wall (green, cyan and magenta) is mainly composed of bulk beryllium, beryllium coated or tungsten coated tiles. The coated tiles are either made of CFC or Inconel. The divertor (red and purple) is composed of bulk tungsten or CFC tungsten coated tiles. The details of the tile assembly are shown for the wide outer poloidal limiter, the IWGL and the divertor horizontal, outer target (bulk W).

temperature ( $T_{\text{melt}} = 3422^\circ\text{C}$ ) but is rather brittle and gets weaker under thermal stresses (thermal fatigue). A thermal cycle above  $1200^\circ\text{C}$  must be budgeted to allow for a reasonable lifetime of the components [1]. In JET only the divertor outer, horizontal target is made of bulk W. The remaining PFCs in the divertor are made of CFC W coated tiles. Previous R and D activities showed that the risk of delamination of the coating is very small if the temperature is kept below  $1200^\circ$ . The design of the ILW was mainly driven to optimize the PFC power handling capability, thereby minimizing the constraints on the plasma operations. There is one major difference in JET compared to ITER: the JET PFCs are all inertially cooled.

The PFC power handling capability, constrained by material limits, depends on the expected power loads from the plasma flux. Given the total power entering the scrape-off layer (SOL),  $P_{\text{SOL}}$ , one can in principle predict the power distribution,  $q_{\text{PFC}}$  ( $\text{MW m}^{-2}$ ), and thereby the peak heat load,  $q_{\text{pk}}$  ( $\text{MW m}^{-2}$ ), onto the PFCs. The expected power load distribution is usually predicted assuming a known parallel heat flux profile in the SOL,  $q_{\parallel}(r_{\text{mid}})$ , defined at the outer mid-plane ( $r_{\text{mid}}$  is the outer mid-plane radius). It is assumed that heat perpendicular transport is poloidally symmetric and dominated by diffusion such that  $q_{\parallel}(r_{\text{mid}}) = q_{0\parallel} \exp[-r_{\text{mid}}/\lambda_q]$ , where  $\lambda_q = q/\nabla q$  is the SOL power decay length. The properties at the outer mid-plane (upstream) are then projected along the field lines down to the PFCs assuming toroidal symmetry.

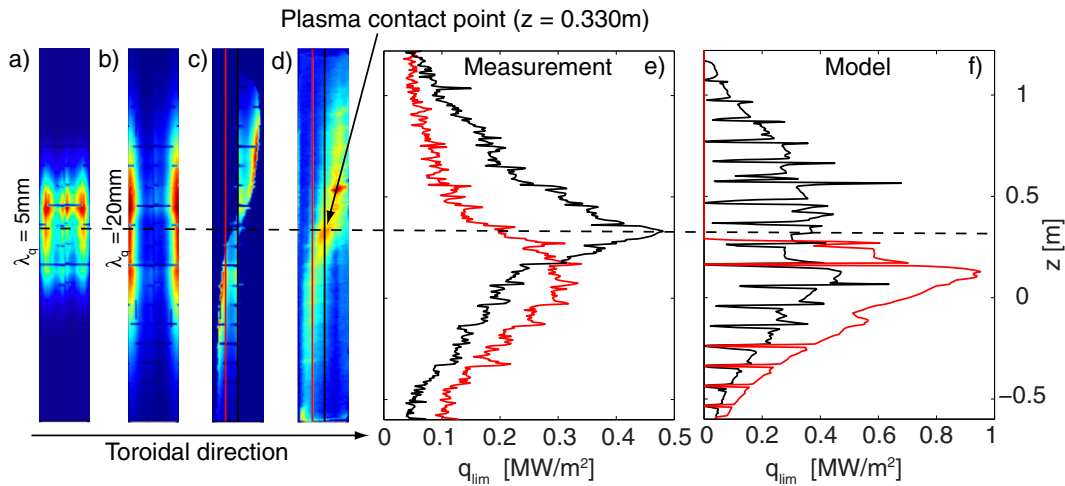
An important feature of the PFC design is to avoid leading edges. The field line angles onto the PFC surface are usually rather shallow, typically  $80^\circ < \theta_n < 89^\circ$ , where  $\cos(\theta_n) = |\mathbf{B} \cdot \hat{\mathbf{n}}|$  with  $\mathbf{B}$ , the magnetic field and  $\hat{\mathbf{n}}$ , the normal vector to the PFC surface. The heat load onto the PFC surface is determined by:  $q_{\text{PFC, surf}} = \cos(\theta_n) q_{\parallel, \text{PFC}}$ , where  $q_{\parallel, \text{PFC}}$  is the parallel heat flux at the PFC location. Any gap between surfaces potentially have field lines reaching an edge orthogonal to the surface, which yields  $q_{\text{PFC, edge}} = \sin(\theta_n) q_{\parallel, \text{PFC}}$ . The heat loads onto a leading edge can therefore be much higher than on the surface:  $10 < q_{\text{PFC, edge}}/q_{\text{PFC, surf}} = \tan(\theta_n) < 30$ . Particular attention has been paid in the design in order to shadow any leading edges.

This paper will review the different models used to predict the heat load distribution onto the PFCs and compare with the measurements made in dedicated experiments. In section 2, we discuss the power handling of the first wall and why the protection system of the ILW (PIW) did not prevent melt events when we challenged its power handling capability. In section 3, we discuss the power handling capability of the bulk W. The CFC W coated components have not received significant heat loads during the last experimental campaign and inspections revealed a few, very localized, signs of delamination but nothing systematic.

## 2. Beryllium first wall

The first wall is composed of castellated Be tiles [2], highlighted in green in figure 1. Other components of the first wall are made of Inconel or CFC with a coating of either beryllium (cyan) or tungsten (magenta), but will not be discussed in this paper as they are barely exposed to plasma loads (they are recessed with respect to the bulk Be poloidal limiters by at least 40 mm). The inner and outer wall is composed of discrete, unevenly distributed poloidal limiters. The outer poloidal limiters (OPL) and inner wall guard limiters (IWGL) are composed of 24 and 19 tiles respectively. The detailed tile assemblies of the IWGL and wide OPL are shown in figure 1. These are the key PFCs of the first wall and we will focus mainly on the inner limiter since this is where the plasma starts up and terminates most of the time. The first wall at the top of the machine is composed of 64 rows of 8, roof shaped tiles (upper dump plates) and of 20 pair of cylindrically shaped tiles (upper inner protections). These will be discussed at the end of this section as they are important during transient power loads resulting from uncontrolled plasma terminations (disruptions).

The inner wall tiles were designed to minimize the peak heat load onto the limiter for SOL power decay length in the range:  $5 \leq \lambda_q \leq 20 \text{ mm}$  (defined at the outer mid-plane). A map of the heat flux deposited onto the IWGL 8Z, predicted by the three-dimensional field line tracing and heat flux calculation code PFCFLUX [3], is shown in figures 2(a) and (b) for  $\lambda_q = 5$  and 20 mm, respectively.



**Figure 2.** Heat flux map on the IWGL predicted with PFCFLUX without shadowing for a power decay length,  $\lambda_q = 5$  mm (a) and  $\lambda_q = 20$  mm (b). (c) The same heat flux map as (b) but with shadowing from other PFCs taken into account. (d) Heat flux map derived from IR thermography. (e), (f)  $q_{\text{lim}}$  poloidal profiles at the centre ( $\phi = 1.06$ ) and on the left wing ( $\phi = 1.04$ ) from the simulated map (c) and measured map (d).

In both cases the magnetic equilibrium comes from the same experiment—JET pulse number (JPN) 80836—in which  $P_{\text{SOL}} = 1.54$  MW. It clearly shows that a shorter  $\lambda_q$  concentrates the heat load closer to the plasma contact point. In both cases the peak heat load remains located near the wings of the limiter but the profile is flatter when  $\lambda_q = 5$  mm. A realistic prediction of the heat load distribution can only be obtained if one takes into account the shadowed areas (in dark blue in figure 2(c)) from the other limiters. By definition in the code a shadowed area receives no heat flux if it is not connected to the outer mid-plane. In our specific case, an area is considered wetted if the field lines, starting from the limiter, do not intersect another PFC after 30 m (the connection length between the contact point and the outer mid-plane is 25 m). This is of course a simplified view and a comparison with the measurements (figure 2(d)), derived from the images taken with a wide angle, medium wavelength infrared (MWIR,  $4 \mu\text{m}$ ) camera [4], shows that the reality is slightly more complex than what the model predicts. A comparison between the measurements and the model of two poloidal (vertical) profiles (figures 2(e) and (f), respectively) leads to one conclusion: the measured peak value ( $q_{\text{pk, meas}} \simeq 0.5 \text{ MW m}^{-2}$  on the ridge) is about two times lower than that predicted by the code ( $q_{\text{pk, pred}} \simeq 1 \text{ MW m}^{-2}$  on the wing) and is not at the same location. Our model is therefore rather conservative since it overestimates the peak heat load. The fact that we measure the peak heat load onto the limiter ridge, exactly at the plasma contact point rather than on the wings is not yet fully understood and is discussed in detail in [5]. Note that  $q_{\parallel}(r_{\text{mid}})$  inferred from the IR measurements is such that  $q/\nabla q$  is not constant, with a much steeper gradient in the near SOL. This means that the SOL width cannot be described with a single  $\lambda_q$ . On the OPL the difference between the model and the measurements is smaller and absolute values of heat loads are in much better agreement.

The inner wall was challenged during a dedicated experiment (JPN 83620) where  $P_{\text{SOL}} = 4.7$  MW (to be compared with 1.54 MW in our previous example), applied for about 8 s using 5 MW of neutral beam heating power.

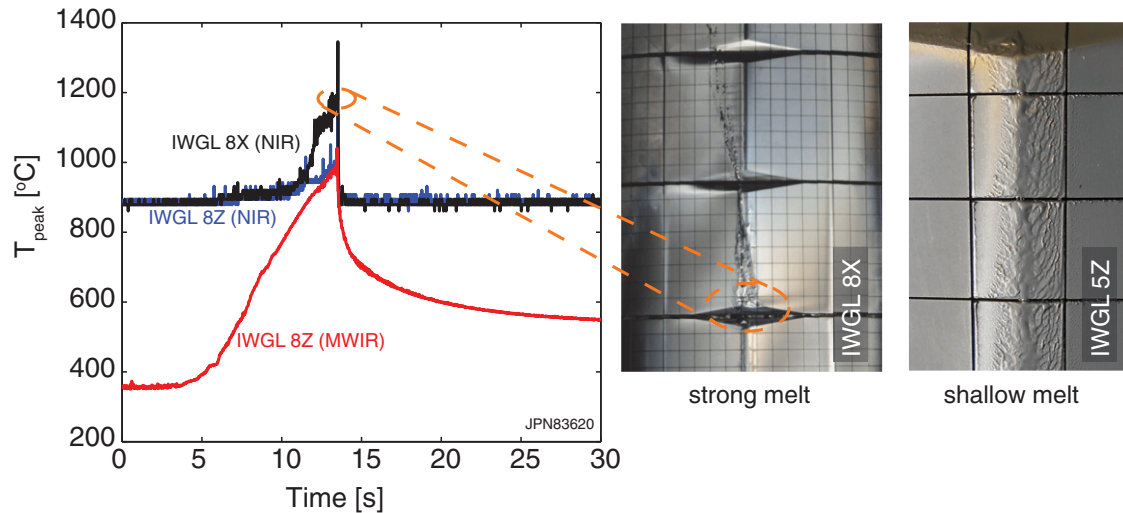
This experiment relied on the PIW which safely stops the plasma pulse if an overheating of the wall is detected [6]. The peak heat load measured on the IWGL 8Z is  $q_{\text{pk, meas}} = 3 \text{ MW m}^{-2}$ , leading to a maximum  $T_{\text{peak}} \simeq 1000^\circ\text{C}$  (see MWIR measurement in figure 3), far from the melt temperature, even taking into account uncertainties ( $\pm 100^\circ\text{C}$  at  $1000^\circ\text{C}$ ). However, an inspection showed strong melts on the ridge of the neighbour limiter (IWGL 8X) (see picture in figure 3) [7], which coincides with the plasma contact point. A comparison of  $T_{\text{peak}}$  measured on the IWGL 8Z and 8X using the protection cameras (NIR) indicated that there was a strong toroidal asymmetry of the power load onto the limiter. Using  $T_{\text{peak}}$  from the NIR camera we estimate that  $q_{\text{pk}} \simeq 6 \text{ MW m}^{-2}$  on the IWGL 8X. This could be explained by a 1 mm misalignment of the limiters if  $q/\nabla q = 5$  mm in the near SOL, which is realistic. A systematic inspection of the inner wall revealed that four limiters were strongly melted, four limiters had shallow melts (see example in figure 3) and eight limiters were intact. The protection system did not prevent melting because it was monitoring the IWGL 8Z only and assumed toroidal symmetry. In the next campaign the PIW cameras will monitor four IWGLs.

During disruptions, the amount of energy (thermal and magnetic) dissipated by radiation is about two times lower with the ILW (10% of the total energy in average) than with the JET carbon wall [8]. The energy driven by the plasma to the wall is therefore much higher and led to systematic melts on the 96 rows of the upper dump plate and upper inner protections. Mitigation of these thermal loads by massive gas injection of a mixture of deuterium and argon is now mandatory for plasmas operating at plasma current  $I_p \geq 2.0$  MA.

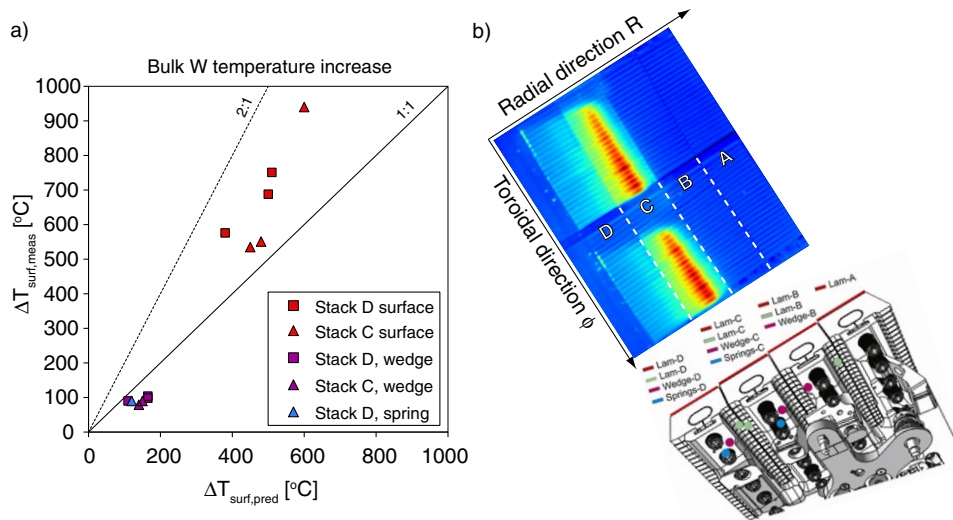
### 3. Tungsten divertor

The divertor horizontal, outer target is composed of 48 pairs of tiles. Each tile assembly (figure 1) is composed of four stacks, each stack being made of 24 lamellas approximately 6 mm wide (toroidal direction), 60 mm long (poloidal direction) and





**Figure 3.** (a) Peak temperature,  $T_{\text{peak}}$  measured on the IWGL 8Z by the MWIR camera (red), by the PIW (NIR,  $1 \mu\text{m}$ ) camera (blue), and on the IWGL 8X by the PIW camera (black). (b) Picture of the melt damage that occurred during JPN 83620 on the IWGL 8X. The melt damage coincides with the plasma contact point.



**Figure 4.** (a) Temperature increase measured in the bulk W tile as a function of that predicted by the GTM code. Triangles and squares indicate which stack the measurement comes from. The colours indicate where the measurement has been taken: on the surface,  $T_{\text{surf}}$  (red), at the wedge,  $T_{\text{wedge}}$  (purple), or on the springs,  $T_{\text{spring}}$  (blue). (b) Illustration of the instrumentation of the tile for temperature measurement.  $T_{\text{surf}}$  can be measured at any point on the tile using IR images.  $T_{\text{wedge}}$  and  $T_{\text{spring}}$  are measured with thermocouples at the position indicated by the purple and blue dots respectively.

40 mm deep. Compared to the former CFC outer target, where each tile was made of a single block, this design is less efficient for diffusing the heat into the tile volume. Since the plasma heat load is mainly localized on one stack, the heat diffuses down to the tile carrier much more quickly. The tile carrier is made of Inconel which has a temperature limit of  $600^\circ\text{C}$ . The lamellas are mounted on the Inconel wedges and pre-loaded using springs at the back of the wedges, which have a temperature limit of  $330^\circ\text{C}$  and are therefore the weakest part of the tile assembly. According to the GTM model, which has been validated on the Marion facility [9], this limit imposes that no more than 60 MJ can fall on a stack, which typically corresponds to a total energy onto the outer target of 74 MJ and a total energy into the SOL of 111 MJ (assuming a divertor power load asymmetry:  $P_{\text{div, in}}/P_{\text{div, out}} = 1/2$ ) [1]. This is too restrictive from the plasma scenario point

of view and a strike point sweeping is necessary to spread the heat load over more than one stack. This method has been used very successfully in the last JET campaigns. Note that the limit on the divertor target (energy density) contrasts with the limit on the Be wall, which is determined by the peak power density. Although the Be wall is also inertially cooled, the surface temperature of the Be PFCs would reach the melting limit well before the energy load became a problem for the tile carrier.

The predictions of the GTM model have been compared with measurements in dedicated experiments with moderate input power ( $P_{\text{heat}} \leq 15 \text{ MW}$ ). The measurements of the tile surface temperature were made using IR thermography [10, 11] and the temperature at the back of the lamellas, on the wedges (tile carrier) and on the springs was measured by thermocouples (figure 4(b)). The measured

temperature increase,  $\Delta T_{\text{meas}}$  is compared to the GTM predictions,  $\Delta T_{\text{pred}}$ , in figure 4(a) for different stacks (C and D) and different measurement positions. It shows that  $\Delta T_{\text{meas}}$  at the surface is higher by up to 50% than that predicted. On the other hand,  $\Delta T_{\text{meas}}$  in the carrier and spring are lower. One could conclude that the diffusion to the tile carrier is less efficient than expected (lower thermal contact), which is positive in terms of the tile carrier protection. However, this indicates that operations are slightly more restricted if one wants to avoid thermal fatigue, since high surface temperatures are reached more quickly than expected. Note that for the GTM model, the global wetted fraction  $\text{GWF} = 1$ , whereas in reality  $0.7 \leq \text{GWF} \leq 0.8$  (depending on the safety factor) as illustrated by the IR image in figure 4(b). This could also explain the higher measured  $\Delta T_{\text{surf}}$  since a lower GWF means a higher peak heat load for the same  $P_{\text{SOL}}$ . This will be verified with deeper analysis of our measurements. The IR image in figure 4 shows that the leading edges are properly shadowed. Slight non-uniformity of the heat load on lamellas is the result of small vertical misalignment ( $\Delta z \leq 120 \mu\text{m}$ ), which is well below the tolerance of the design ( $\Delta z_{\text{max}} = 400 \mu\text{m}$ ). Overall, the divertor target went about 300 times above  $1200^\circ\text{C}$  and no sign of cracks at the surface of the lamella (that thermal fatigue would eventually lead to) has been observed.

#### 4. Conclusion

The power handing capability of the JET ILW has been tested experimentally and met the expected performance despite small differences between the measurements and the predictions of our models. The model used for the first wall proved to be robust since it overestimated the peak power load. The measured peak power load systematically coincides with the plasma contact point, which is not predicted by the model. This cannot be a measurement artefact since it is where molten beryllium was observed when the wall was challenged at high power. The molten Be revealed that the power load is not toroidally symmetric, and this must be taken into account in the first wall protection strategy. The measurements in the divertor indicate that the thermal contact between the tungsten and the tile carrier is lower than that in the model, leading to higher surface temperature than expected. Surface temperature measurement on both the first wall and the divertor showed that the leading edges are properly shadowed.

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